AMENDMENT

In the Claims:

This listing of claims will serve to replace all prior versions and listings of claims in the present application:

- (Currently amended) A rotary liquefied natural gas boil-off compressor comprising having at least two compression stages in series, a gas passage passing through the series of compression stages, the gas passage extending through and being in heat exchange relationship with at least one cooling means disposed between the ereach pair of compression stages, characterised in that wherein the cooling means or at least one of the cooling means is a cryogenic cooling means and in that there is having valve means for controlling flow of cryogenic coolant into the cryogenic cooling means in response to the an inlet temperature, or a related parameter, of the next compression stage next in series downstream of the cryogenic cooling means so as, in use, to maintain said inlet temperature at a chosen sub-ambient temperature or between chosen sub-ambient temperature limits.
- (Currently amended) The compressor according to claim 1, characterised in that wherein the cryogenic cooling means comprises an indirect cooling means.
- (Currently amended) The compressor according to claim 1, characterised in that wherein the cryogenic cooling means comprises a direct cooling means.
- (Currently amended) The compressor according to claim 3, characterised in that wherein the direct cooling means comprises a chamber having an inlet for the introduction of a cryogenic liquid.

- (Currently amended) The compressor according to claim 4, characterised in that the outlet of wherein the direct cooling communicates means comprises an outlet in communication with a vessel adapted to disengage particles of liquid from the natural gas, the vessel having an outlet for the natural gas communicating to communicate with said next compression stage.
- 6. (Currently amended) The compressor according to claim 1, characterised in that there is further comprising a cryogenic cooling means intermediate each pair of successive compression stages.
- 7. (Currently amended) The compressor according to claim 1, characterised in that wherein there are at least three compression stages in sequence, and in that there is at least one direct cryogenic cooling means and at least one indirect cryogenic cooling means.
- 8. (Currently amended) The compressor according to claim 7, characterised in that wherein an inlet of a the at least one direct cryogenic cooling means communicates with an outlet of an the at least one indirect cooling cryogenic means.
- (Currently amended) The compressor according to claim 1, characterised in that there is comprising a cryogenic cooling means downstream of the <u>a</u> final <u>stage of the series of</u> compression <u>stagestages</u>.
- 10. (Currently amended) The compressor according to claim 1, eharacterised in that there is comprising a cryogenic cooling means upstream of the a first stage of the series of compression stagestages.

- 11. (Currently amended) The compressor according to claim 1, eharacterised in that wherein the compressor has comprises an intermediate inlet communicating with a forced liquefied natural gas vaporiser.
- 12. (Currently amended) A liquefied natural gas storage tank having an outlet for boiled-off natural gas communicating with a the compressor as claimed in of claim 1, the said cryogenic cooling means communicating in communication with the liquefied natural gas in the storage tank.
- 13. (Currently amended) A method of operating a rotary liquefied natural gas boil-off compressor having at least two compression stages in series and a gas passage passing through the series of compression stages, the method comprising cooling [[the]] compressed boiled-off natural gas by means of a cryogenic coolant downstream of one of the compression stages and upstream of another one of the compression stages in series, monitoring [[the]] an inlet temperature, or a related parameter, of the compressed natural gas at [[the]] an inlet to the other compression stage, and adjusting [[the]] a flow rate of the cryogenic coolant se as to maintain said inlet temperature at a chosen subambient temperature or between chosen sub-ambient temperature limits.
- 14. (Currently amended) The method according to claim 13, characterisedin that wherein the inlet temperature of each of the compression stage stages is maintained at a temperature in the range of minus 50 °C to minus 140°C.
- 15. (Currently amended) The method according to claim 14, characterisedin-that wherein the pressure ratio across each of the compression stage stages is in the range 2.15: 1 to 3: 1.

16. (Currently amended) The method according to claim [[15]] 14, characterised in that wherein the pressure ratio across each of the compression stagestages is in the range 2.5: 1 to 3:1.